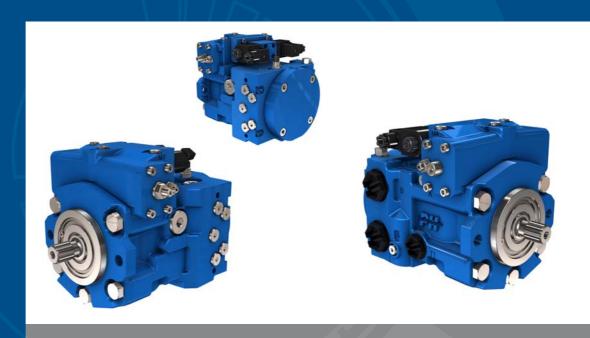
# PM50 VARIABLE DISPLACEMENT PUMP CLOSED LOOP CIRCUIT



TECHNICAL CATALOG



### **OVERVIEW**

PM50 is a variable displacement, axial piston pump, with swashplate system, for closed loop hydrostatic transmissions.

It provides a continuously variable flow rate between zero and maximum in forward and reverse direction. Flow rate is proportional to rotation speed and swashplate angle.

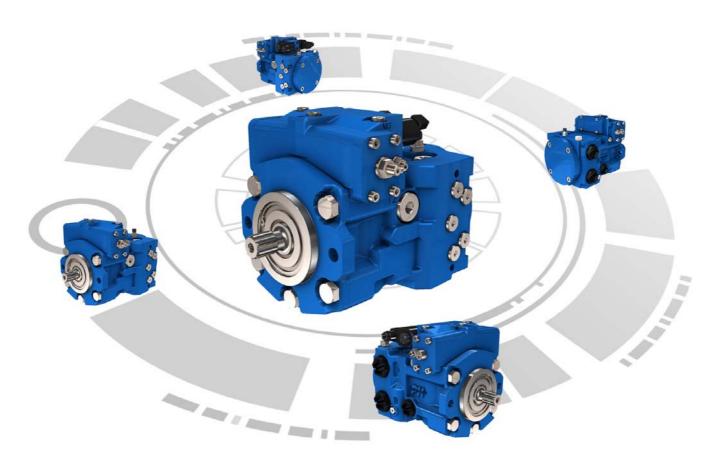
It can feature a charge pump to keep the circuit pressurised. This avoids risk of cavitations and ensures a good performance of the transmission.

It offers several types of control: Servo mechanical, Servo hydraulic, Electrical, Electro-proportional and Automotive. Hydraulic and Electro-proportional ones can be equipped with feed-back device.

It is equipped with high pressure relief valves and can be delivered with auxiliary gear pumps.

It is available in single or tandem versions.

As options, PM50 can be featured with flushing valve and filter on charge pressure line.



		PM50-40	PM50-45	PM50-52	
Displacement	cm³/rev [in³/rev.]	40 [2.44]	45 [2.75]	52 [3.17]	
Theoretical Flow at rated speed	L/min [GPM]	144 [38.04]	162 [ <i>4</i> 2.79]	187,2 <i>[49.45]</i>	
Max. Theoretical absorbed power at 320 bar [4 641 PSI]	kW [hp]	76,8 [103]	86,4 [116]	99,8 <i>[134]</i>	
Theorical absorbed torque at 100 bar [1 450 PSI]	N.m [in.lbf]	63,7 [564]	71,7 [635]	82,8 [733]	
Moment of inertia	kg.m² [slug.ft²]		0.0054 [0.0038]		
Mounting flange			SAE B, SAE BB		
Controls		Servo mechanical, Servo hydraulic, Electrical, Electro-proportional, Automotive			
Mass	kg [lb]	32 [70.5] with servo control S			
Rotation		Clockwise or Counterclockwise			

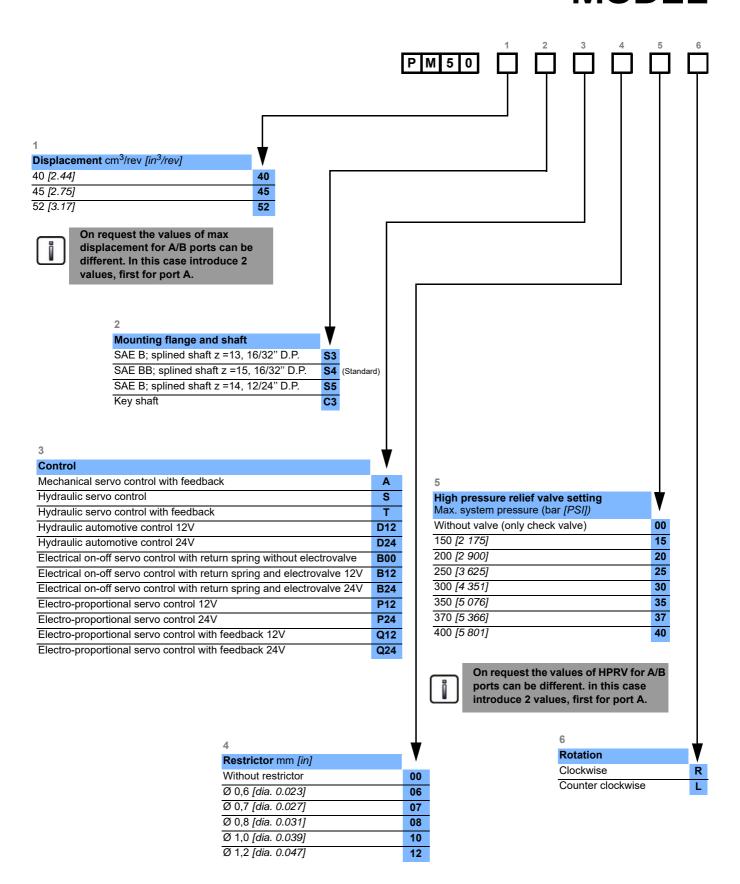




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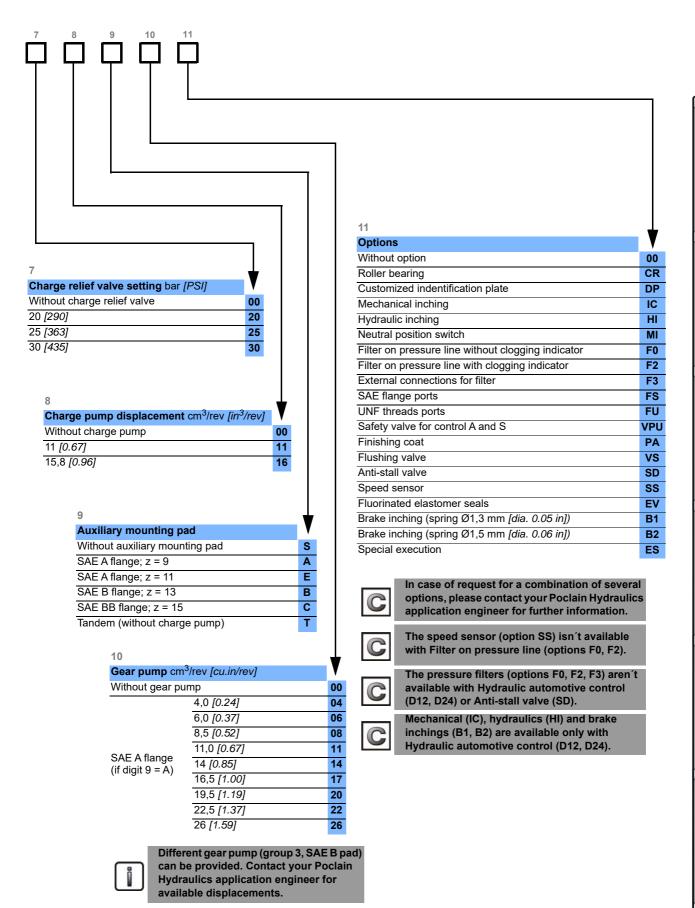
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### **MODEL**





### CODE



Technical pecifications

Operating Parameters

System design

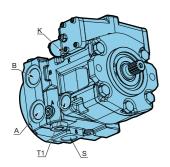
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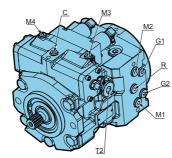
controls

ptions

## **TECHNICAL**

### Port characteristics





Port	Function	ISO 1179-1 (standard)	Maximum lenght of nipples [mm] <i>[in]</i>
A/B	Services	34G-G1	
С	Case pressure	13G-G1/4	
G1/G2	Auxiliary/Charge pressure	13G-G1/4	
M1/M2	A/B pressure	10G-G1/8	
M3/M4	Servo control	13G-G1/4	12,5 <i>[0.49]</i>
K	External servo pilot	10G-G1/8	
R	Servo pilot pressure	13G-G1/4	
S	Suction	34G-G1	
T1/T2	Drain	27G-G3/4	

See options FS and FU on page 41 for SAE flange and UNF port size.  $\,$ 



Model Code

Technical specifications

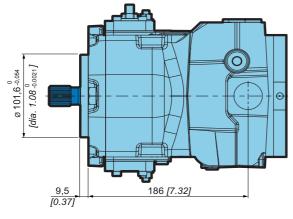
Operating Parameters

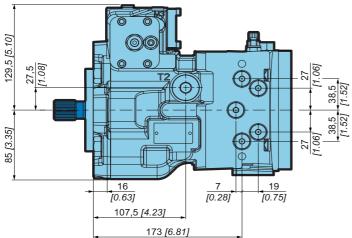
### **SPECIFICATIONS**

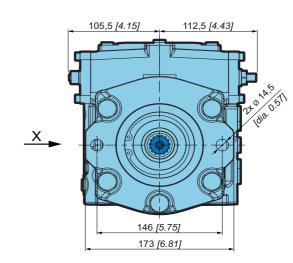
### **Main dimensions**

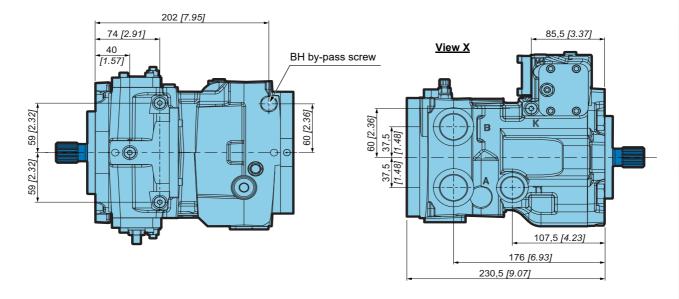
PM50 - SAE BB - splined shaft with hydraulic servo control with internal charge pump and without auxiliary mounting pad











### **OPERATING PARAMETERS**

### **Operating parameters**

			PM50-40	PM50-45	PM50-52
0	Minimum			700	
Speed ratings	Max. without load	min <sup>-1</sup> (rpm)		3 600	
atiligs	Max. with load	<u> </u>		3 400	
	Rated			300 [4 351]	
System	Maximum	bar [PSI]		400 [5 801]	
ressure	Minimum low loop	<del></del>		15 [218]	
	Mini continuous	bar (abs.)		0,8 [11.6]	
llet pressure	Mini (cold start)	[PSI abs.]		0,5 [7.2]	
ase	Continuous	bar <i>[PSI]</i> _		2 [29]	
ressure	Maximum (cold start)	— Dai [F31] —		3,5 [50.7]	
harge	Standard version			25 [362.6]	
ressure	Max. charge pressure	—— bar <i>[PSI]</i> —		30 [435]	
Servo case pressure	Maximum	bar [PSI]		30 [435]	

### Charge pressure

A charge flow is required to maintain a positive pressure in the low pressure loop of a closed loop hydrostatic transmission. Charge pressure ensures proper lubrication and rotating group operation. It is recommended to maintain the charge pressure at a minimum of 15 bar [218 PSI] above case pressure. For more details, refer to charge pump paragraph, page 18.

### Case pressure

Case pressure must be maintained within the limits shown in the table "Operating parameters". Ensure housing is always filled with hydraulic fluid and especially during start-up of the machine.

### **Pressure ratings**

### Maximum peak pressure

It is the maximum allowable pressure. It is equivalent to the maximum setting of the maximum high pressure relief valve. A self-propelled machine can reach the maximum peak pressure value no more than 1-2% of that work cycle.

### Work cycle

A fundamental factor for ensuring correct hydrostatic transmission sizing is the machine work cycle (pressure-time ratio, seasonality, pressure vs. percentage of time at max. displacement, machine type). Part service life depends on the correct choice in relation to the work cycle.

### Overloads

It is mandatory to protect parts against any possible overloads.

### **Speed ratings**

The table "Operating parameters" gives minimum and maximum rated speeds. Note that all displacements might operate under different speed limits. Definitions of these speed limits appear below.

**Maximum speed** is the highest operating speed allowed. Over speeding reduces pump life time, can lead to loss of hydrostatic power and braking capacity. Never exceed the maximum speed limit under any operating conditions.

Nominal speed is the speed offering the maximal efficiency.

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### Inlet pressure

Charge pump inlet pressure is key for acceptable pump life and performances. A continuous inlet pressure of not less than 0,8 bar abs. [11.6 PSI abs.] is recommended. A continuous inlet pressure less than 0.5 bar abs. [7.2 PSI abs.] indicates inadequate inlet design or a restricted filter. Pressures less than 0.5 bar abs. [7.2 PSI abs.] can happen at cold start, but should increase with oil temperature.

### Theoretical output

Theoretical output flow is a function of pump displacement and speed. It is relevant to size the rest of the circuit. Theoretical flow does not take into account losses due to leakage or variations in displacement. Refer to performances, page 6, for volumetric and overall efficiencies at various operating speeds and pressures.

### Poclain Hydraulics recommandations for fluid



Poclain hydraulics recommends the use of hydraulic fluids defined by the ISO 15380 and ISO 6743-4 standards. For temperate climates, the following types are recommended.

- HM 46 or HM 68 for fixed installations.
- HV 46 or HV 68 for mobile installations.
- HEES 46 for mobile installations.

These specifications correspond to category 91H of the CETOP standard, parts 1, 2 and 3 of the DIN 51524 standard, and grades VG32, VG 46 and VG68 of the ISO 6743-4 standards.



It is also possible to use ATF, HD, HFB, HFC or HFD type hydraulic fluid upon Poclain Hydraulics specific approval of the components' operating conditions.

Standardized designations for the fluids

- **HM**: Mineral fluids having specific antioxidant, anticorrosion and antiwear properties (HLP equivalent to DIN 51524 parts 1 and 2).
- HV: HM mineral fluids providing improved temperature and viscosity properties (DIN 51524 part 3).
- HEES: Biodegradable fluids based on organic esters.



It is also possible to use a fluid that meets the biodegradability criteria and is compatible in the event of accidental food contact. The BIOHYDRAN FG 46 fluid designed by the company Total has undergone testing of its properties and performance on our test benches. Since this type of fluid has not yet been categorized, it is the responsibility of machine manufacturers to validate its compatibility with all of the components used in order to guarantee that the intended functions will be fulfilled and this for the desired life time of all equipment items.



For biodegradable fluids, consult your Poclain Hydraulics' application engineer



During operation, the temperature of the oil must be between 0°C [32°F] and 80°C [176°F]; the minimum and maximum temperatures may be exceeded momentarily by ± 20°C [± 68°F] for a duration of less than 30 minutes. For all applications outside these limits, please consult with your Poclain Hydraulics' application engineer.

### Pump storage



If the pump stays on stock for more than 6 months, a status verification must be performed before you install it on a machine. Pay attention to sealing condition, rust presence and free rotation of shaft.

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### Fluid and filtration

The contaminating particles suspended in the hydraulic fluid cause the hydraulic mechanisms moving part wear. On hydraulic pumps, these parts operate with very small dimensional tolerances. In order to reach the part life, it is recommended to use a filter that maintains the hydraulic fluid contamination class at a max. of:

9 according to NAS 1638 20/18/13 according to ISO 4406:2021

According to the type of application decided for the pump, it is necessary to use filtration elements with a filtration ratio of:

 $\beta$  20 to 30 ≥ 100

Making sure that this ratio does not worsen together with the increasing of the filter cartridge differential pressure.

If these values cannot be observed, the component life will consequently be reduced and it is recommended to contact the Poclain Hydraulics Customer Service.

### Filters on charge circuit

Filters on the charge circuit (F0-F2) are designed without by-pass. The max. pressure drop on the filtration part must not exceed 2 bar [29 PSI] (3 bar [43.5 PSI] in case of cold starting) at pump full rating. To monitor the pressure drop, It is recommended to use the clogging indicator on the filtration element (F2 option). Contact your Poclain Hydraulics Application engineer, each time the pump is not charged by its internal charge pump.

Filters on charge circuit are mounted on the pump special support.

### Filters assembling

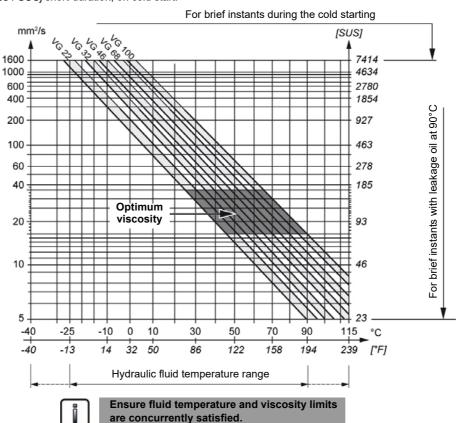
The suction filter is mounted on the suction line. Check that the pressure before the charge pump is 0.8 bar abs. [11.6 PSI abs.], measured on the pump suction port (0.5 bar [7.2 PSI] for cold starting).

### Viscosity range

For both max, efficiency and life of the unit, the operative viscosity should be chosen within the optimum range of:  $\sqrt{\text{opt}} = \text{optimum operating viscosity from 16 to 36 mm}^2/\text{s [from 74.1 to 166.8 SUS] referred to the closed loop temperature.}$ 

### Working conditions: the following limits of viscosity apply

 $\sqrt{\text{min}} = 5 \text{ mm}^2/\text{s}$  [23 SUS] short-duration at a max. permissible leakage oil temperature of 90° C [194°F]  $\sqrt{\text{max}} = 1000 \text{ mm}^2/\text{s}$  [4 634 SUS] short-duration, on cold start.



### SYSTEM DESIGN PARAMETERS



Consult your Poclain Hydraulics application engineer to validate your design parameters before using the pump in your application.

### Sizing equations

The following equations are helpful when sizing hydraulic pumps. Generally, the sizing process is initiated by an evaluation of the machine system to determine the required motor speed and torque to perform the necessary work function. First, the motor is sized to transmit the maximum required torque. The pump is then selected as a flow source to achieve the maximum motor speed.

	Output flow Q	$= \frac{V_g.n.\eta_v}{1000}$	(l/min)
SI units	Input torque M	$= \frac{V_g \cdot \Delta_p}{20 \cdot \pi \cdot \eta_m}$	(N.m)
	Input power P	$= \frac{M. n.\pi}{30 000} = \frac{Q.\Delta_p}{600.\eta_t}$	(kW)
	Output flow Q	$= \frac{V_g \cdot n \cdot \eta_v}{231}$	[GPM]
US units	Input torque M	$= \frac{V_g \cdot \Delta_p}{2 \cdot \pi \cdot \eta_m}$	[lbf.in]

 $V_g$ =Displacement per revolution cm<sup>3</sup>/tr [in<sup>3</sup>/rev]  $\Delta p = p_o - p_i$  (system pressure) bar [PSI]

n = Speed min<sup>-1</sup> [rpm]

 $\eta_V$  = Volumetric efficiency

 $\eta_m$  = Mechanical efficiency

 $η_t$  = Overall efficiency (ηv.ηm)

### Redundant braking system requirement



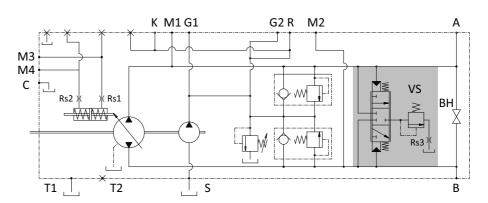
Unintended vehicle or machine movement hazard.

The loss of hydrostatic drive line power, in any mode of operation (forward, neutral, or reverse) may cause the system to lose hydrostatic braking capacity. You must provide a braking system, redundant to the hydrostatic transmission, sufficient to stop and hold the vehicle or machine in the event of hydrostatic drive power loss.

### **Loop flushing**

Closed circuit may require a flushing valve to meet temperature and cleanliness requirements. A flushing valve takes a part of hot fluid flow from the low pressure loop of the system loop for cooling and filtering. Make sure that the charge pump provides adequate flow for the flushing valve flushing and the flushing valve does not cause charge pressure to drop below recommended limits.

See option VS page 45 for more information.



### Reservoir

The reservoir provides clean fluid, dissipates heat, and removes entrained air from the hydraulic fluid. It allows for fluid volume changes associated with fluid expansion and cylinder differential volumes. Minimum reservoir capacity depends on the volume needed to perform these functions. Typically, a capacity of one half the charge pump flow (per minute) is satisfactory for a closed reservoir. Open circuit systems sharing a common reservoir require greater fluid capacity.

Locate the reservoir outlet (suction line) near the bottom, allowing clearance for settling foreign particles. Use a 100 - 125  $\mu$ m screen covering the outlet port.

Place the reservoir inlet (return lines) below the lowest expected fluid level, as far away from the outlet as possible.

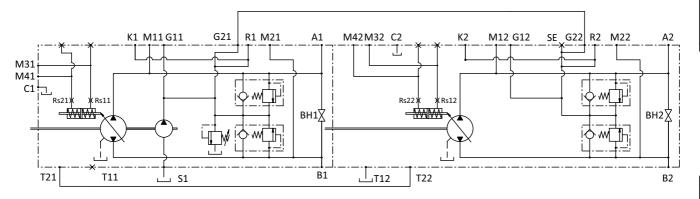
Use a baffle (or baffles) between the reservoir inlet and outlet ports to promote de-aeration and reduce fluid surging.

### Case drain usage for tandem pump

To ensure lubrification of both pumps (with only one charge pump), excess flow from the second pump charge relief valve must be routed into the housing of the first pump.

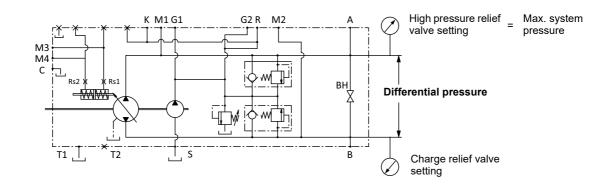


Tandem pumps with the option of opposing port endcaps do not follow the above rule.



### Differential pressure

The differential pressure is the High pressure relief valve setting minus Charge relief valve setting.



### Bearing life and external shaft loading

### Bearing life:

Bearing life is a function of speed, pressure, swashplate angle and external loads. Oil type and viscosity impact bearing life.

	Ball bearing life (B <sub>10</sub> hours)	Roller bearing life (B <sub>10</sub> hours)
PM50-40	21 000	39 000
PM50-45	14 500	27 000
PM50-52	9 500	18 000

#### **Shaft Loads**

Normal bearing life in B<sub>10</sub> hours is shown in the above table. Figures have been calculated under the following operating conditions: A continuous differential pressure of 150 bar *[2 176PSI]*, 1 800 rpm shaft speed, maximum displacement, without any external shaft side load. The data is based on a 50% forward, 50% reverse duty cycle, standard charge pump size, and standard charge pressure.

PM50 pumps are designed with bearings that can accept external radial and thrust loads. The external radial shaft load limits depend on the load position, orientation, and operating conditions of the unit.

The maximum permissible radial load (Re), is based on the maximum external moment (Me), and the distance (L) from the mounting flange to the load. It may be determined using the table and formula below. Thrust (axial) load limits are also shown.

$$Re = Me / L$$

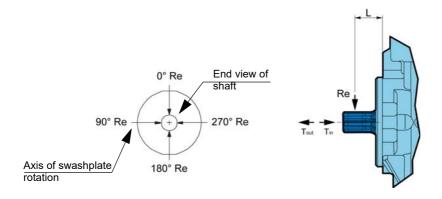
All external shaft loads affect bearing life. In applications with external shaft loads, minimize the impact by positioning the load at 90° or 270° as shown in the figure.

Contact your Poclain Hydraulics representative for an evaluation of unit bearing life if:

- Continuously applied external loads exceed 25 % of the maximum allowable radial load Re.
- The pump swashplate is positioned on one side of center all or most of the time.
- The unit bearing life (B<sub>10</sub>) is critical.

	External moment (Me) N.m [in.lbf] (Based on shaft deflection)	Maximum shaft thrust N [ $lbf$ ] (at $\Delta P$ 180 bar [ $2~611~PSI$ ] and $3~400~rpm$ )
PM50-40	150 [1 328]	1 500 <i>[337]</i>
PM50-45	107 [947]	1 500 [337]
PM50-52	76 <i>[673]</i>	1 500 [337]

### Radial and thrust load position





For an accurate calculation, consult your Poclain Hydraulics application engineer.



### **Hydraulic unit life**

Hydraulic unit life is the life expectancy of the hydraulic components. It depends on speed and system pressure even if , system pressure is the dominant operating variable. High pressure, generated by high load, reduces hydraulic unit life.

Design the hydraulic system according to the expected machine duty cycle. Take in consideration the expected percentages of time at various loads and speeds. Ask your Poclain Hydraulics representative to calculate an appropriate pressure based your hydraulic system design. If duty cycle data is not available, input power and pump displacement are used to calculate system pressure.

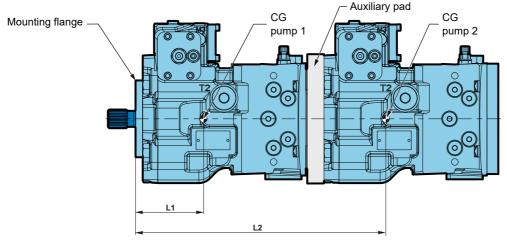
All pressure limits are differential pressures (referenced to charge pressure), taking a normal charge pressure in consideration.

PM50 pumps will meet satisfactory life expectancy if applied within the parameters specified in this technical documentation. For more detailed information on hydraulic unit life see Operating Parameters in page 9.

### **Mounting flange loads**

Adding tandem mounted pumps, and/or tandem auxillary pump(s), subjecting pumps to shock loads may generate excessive loads on the front mounting flange. The overhung load moment for multiple pump mounting can be estimated as shown in the figure bellow

### Overhung load example



For two PM50 in tandem the approximate distances (exact values depend on pumps configuration) of gravity centers from front mounting flange are:

L1 = 92 mm [3.62 inch]

L2 = 330 mm [12.99 inch]

### **Estimating overhung load moments**

W = Weight of pump (kg)

L = Distance from mounting flange to pump center of gravity (CG)

 $M_R = G_R (W_1L_1 + W_2L_2 + ... + W_nL_n)$ 

 $M_S = G_S (W_1L_1 + W_2L_2 + ... + W_nL_n)$ 

### Where:

M<sub>R</sub> = Rated load moment (N.m)

M<sub>S</sub> = Shock load moment (N.m)

GR\*= Rated (vibratory) acceleration (G's) (m/sec²)

G<sub>S</sub>\*= Maximum shock acceleration (G's) (m/sec²)

\*Calculations will be carried out by multiplying the gravity (g = 9.81 m/sec<sup>2</sup>) with a given factor. This factor depends on the application.

Allowable overhung load moment are shown in the above table. Exceeding these values requires additional pump support.

	Rated moment (MR)	Shock load moment (MS)		
	N.m [in.lbf]	N.m [in.lbf]		
PM50-40	900 [7 966]	2 000 [17 701]		
PM50-45 900 [7 966]		2 000 [17 701]		
PM50-52	900 [7 966]	2 000 [17 701]		



For an accurate values and calculations, consult your Poclain Hydraulics application engineer.

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### **FEATURES**

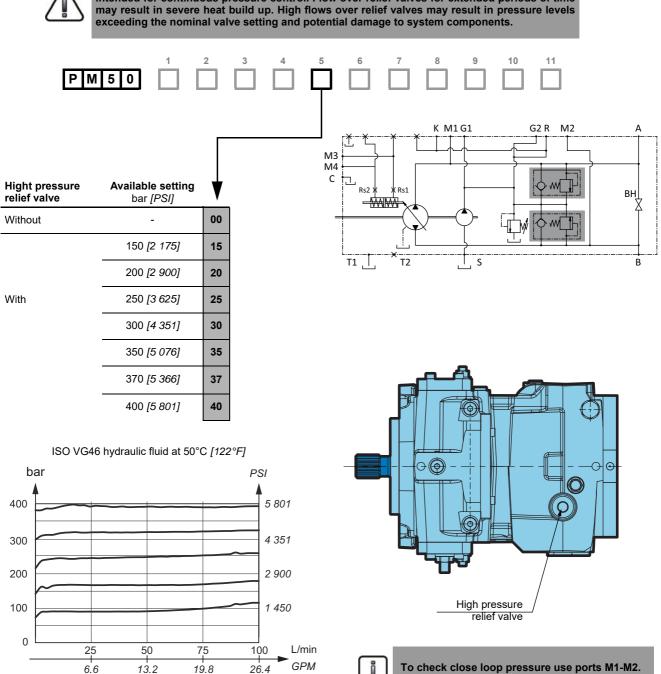
### High pressure relief valve

The High pressure relief valves maintain circuit pressure in the proper range. The check valves allow charge flow to replenish the low pressure loop of the circuit. The high pressure relief valves ensure a high pressure protection of the high pressure loop of the circuit.

High pressure relief valves are available in a large range of settings. They are not adjustable. When high pressure relief valves are not desired, pumps may be equipped with charge circuit check valves only. On request the setting of the max displacement can be different, in this case two values must be indicated in order code (first for port A).



High pressure relief valves are intended for transient overpressure protection and are not intended for continuous pressure control. Flow over relief valves for extended periods of time may result in severe heat build up. High flows over relief valves may result in pressure levels



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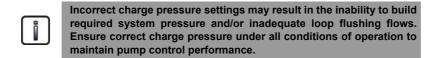
Flow

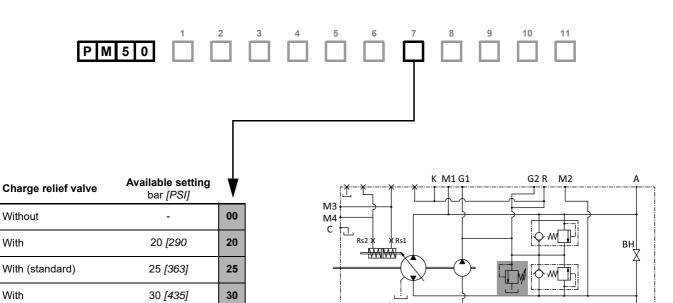


### Charge relief valve

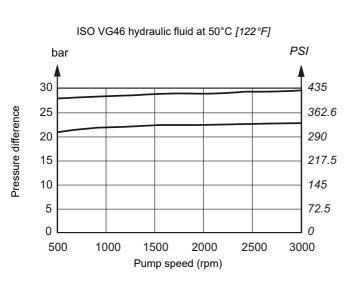
The charge pressure relief valve provides a relief outlet for charge circuit. This valve is used to set the charge pressure of the circuit. Flow through the valve is ported to case.

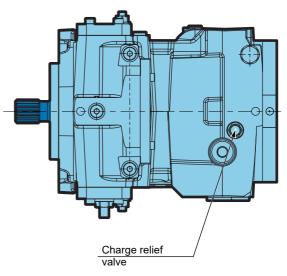
The nominal charge relief setting is referenced to case pressure.





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To check charge pressure use ports G1-G2.

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Technical specifications

**Operating Parameters** 

System design Parameters

Features

Controls

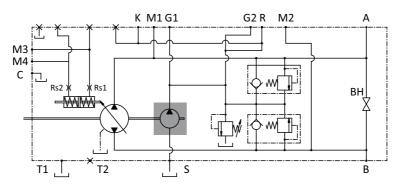
Options

### Charge pump

Charge flow is required on all PM50 pumps used in closed circuit installations. The charge pump provides flow to make up internal leakage, maintain a positive pressure in the main circuit, provide flow for cooling and filtration, replace any leakage losses from external valving or auxiliary systems, and to provide flow and pressure for the control system.

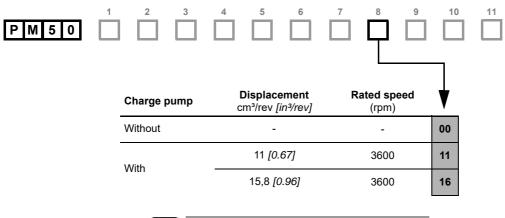
Many factors influence the charge flow requirements. These factors include system pressure, pump speed, pump swashplate angle, type of fluid, temperature, size of heat exchanger, length and size of hydraulic lines, control response characteristics, auxiliary flow requirements, hydrostatic motor type, etc.

Unusual application conditions may require a more detailed review of charge pump sizing. Charge pressure must be maintained at a specified level under all operating conditions to prevent damage to the transmission. Poclain Hydraulics recommends testing under actual operating conditions to verify this.

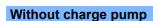


### Charge pump sizing / selection

In most applications a general guideline is that the charge pump displacement should be at least 20% of the main pump displacement.

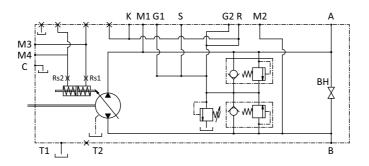


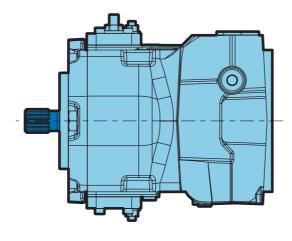
Contact your Poclain Hydraulics application engineer for more information.

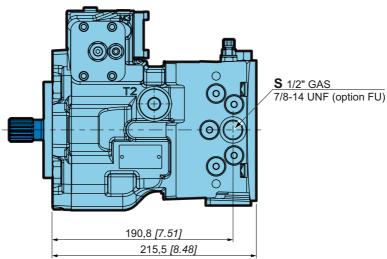




The external charge flow must be the same that the internal charge pump flow and connected with port S. PM50 without internal charge pump is shorter, respect standard with internal charge pump.







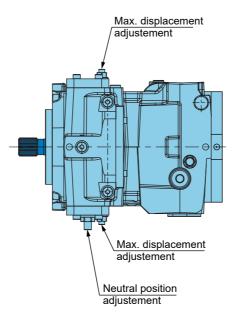
### **Displacement limiters**

PM50 are designed with mechanical displacement (stroke) limiters. You can limit maximum displacement of the pump to a certain per-cent of its maximum displacement to near zero in both direction.

The displacement limiters are located on the both sides of the servo piston and are adjustable by screw. On request the setting of the max. displacements can be different, in this case two values must be indicated in order code (first for port A).

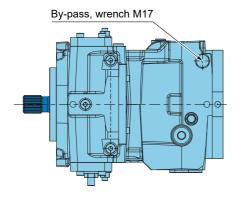


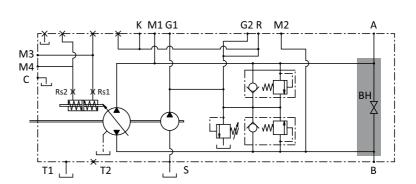
Take care in adjusting displacement limiters to avoid an undesirable condition of output flow or speed. Retorque the sealing lock nut after every adjustment to prevent an unexpected change in output conditions and to prevent external leakage during pump operation.



### **By-pass**

PM50 features a by-pass function. By-passing Port A and Port B is achieved by unscrewing a screw located on the cover. The by-pass connect the ports A-B and must be use only in emergency case and only for short movement.







To avoid leakage, do not exceed two turns of the screw.



By-pass valve is intended for moving a machine for very short distances at very slow speeds. It isn't intended as tow valve.

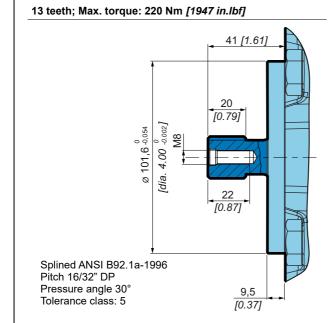


### Mounting flange and shafts

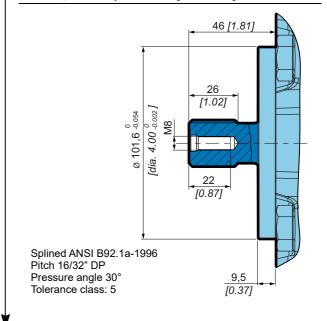


### SAE B - Splined shaft

S4 SAE BB - Splined shaft (standard)



15 teeth; Max. torque: 360 Nm [3186 in.lbf]



### S5 SAE B - Splined shaft

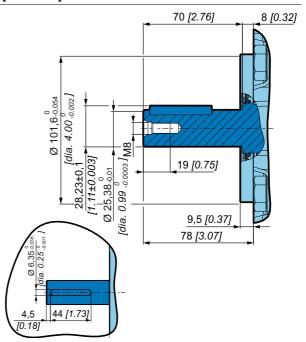
22/04/2024

14 teeth; Max. torque: 600 Nm [5310 in.lbf]

C3 SAE B - Key shaft

56 [2.20] 34,5 [1.36] [dia. 4.00 <sup>0</sup>.002] 0 0 101,6-0,054 22 [0.87] Splined ANSI B92.1a-1996 Pitch 12/24" DP 9,5 Pressure angle 30° [0.37] Tolerance class: 5

 $\emptyset$  = 22,22 mm [dia. 0.87 in]; Max. torque: 220 Nm [1947 in.lbf]



specifications **Technical** 

**Parameters** Operating

System design **Parameters** 

Features

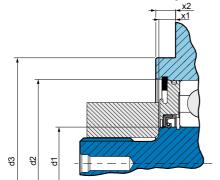
Controls

Options

21

### Fundamental dimensions for coupling assembly

To avoid the contact between rotating and fixed parts the below dimensions for coupling must be observed.

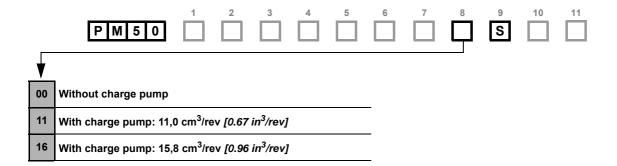


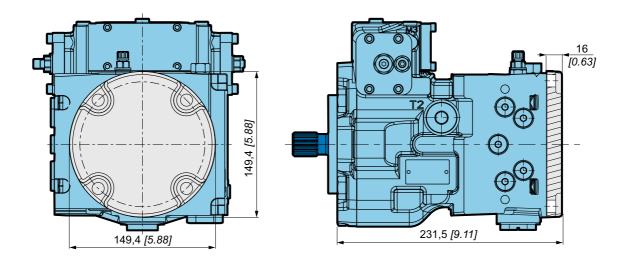
Size	Ød₁	$Ød_2$	$Ød_3$	$\mathbf{X}_{1}$	$\mathbf{X}_{2}$
PM50	35	81 <sup>+0,1</sup>	101,6	8	9,5 <sub>-0,1</sub>
	[1.38]	[3.19 <sup>+0.004</sup> ]	[3.99]	[0.31]	[0.37 <sub>-0.004</sub> ]



For precise info regarding coupling assembly contact your Poclain Hydraulics application engineer.

### **Closed cover**

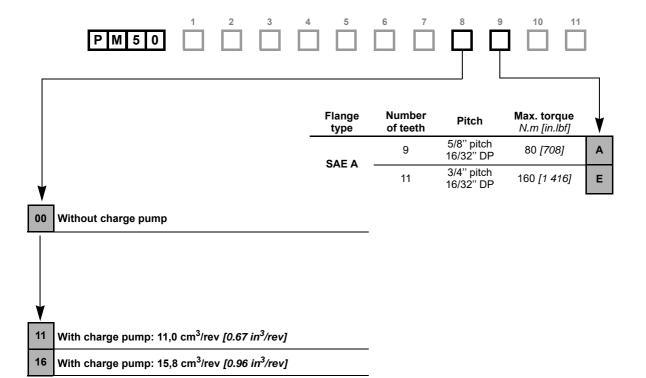




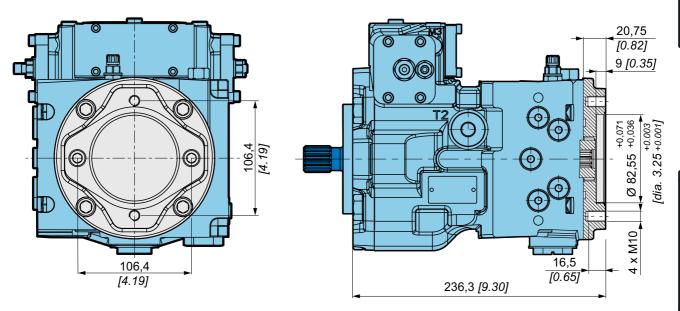


### **Auxiliary mounting pad**

### **SAE A flanges**



Splined ANSI B92.1a-1996 Pressure angle 30° Tolerance class: 5



O-ring: OR-1.78-85.34-NBR70 P/N: A44937U

Î

Do not rotate the thru shaft cover.

, y

specifications

**Technical** 

Operating Parameters

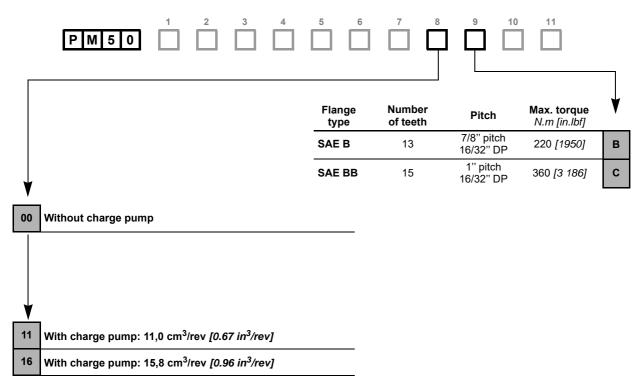
System design Parameters

Features

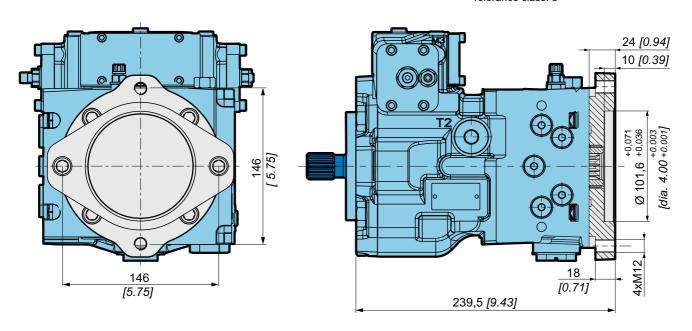
Controls

Options

### SAE-B and SAE-BB flanges



Splined ANSI B92.1a-1996 Pressure angle 30° Tolerance class: 5



O-ring: OR-1.78-101.32-NBR70 P/N: A47888C



Do not rotate the thru shaft cover.

#### **Tandem pumps** Max. torque intermediate coupling 360 N.m [3 186 in.lbf] Front axial Rear axial pump pump M<sub>e1</sub> for the first pump Me input torque $\mathrm{M}_{\mathrm{e}2}$ for the second pump Me3 for the next pump Torque required by auxiliary pumps is additive. Ensure requirements don't exceed shaft torque ratings. Front axial pump Rear axial M 5 pump Number of charge pump **Mounting flange** Charge **Auxiliary mounting** Axial Total axial lenght in the tandem and shaft pump pump flange mm [inch] SAE BB; 15 teeth S4 00 Т **Front** Without Tandem fitting SAE B; 14 teeth S5 S 455,0 [17.91] Without fitting SAE A; 9 teeth Α 475,8 [18.73] 0 charge pump SAE BB; 15 teeth **S4** Without SAE A; 11 teeth Е 00 475,8 [18.73] Rear В SAE B; 13 teeth 479 [18.86] SAE BB; 15 teeth С 479 [18.86] SAE BB; 15 teeth **S4** 11 With т Tandem fitting Front or SAE B; 14 teeth S5 16 Without fitting S 455,0 [17.91] 1 charge pump SAE A; 9 teeth Α 475,8 [18.73] **S4** Е Rear SAE BB; 15 teeth Without 00 SAE A; 11 teeth 475,8 [18.73] SAE B; 13 teeth В 479 [18.86] С SAE BB; 15 teeth 479 [18.86] SAE BB; 15 teeth **S4** 11 SAE BB; 15 teeth С **Front** With or SAE B; 14 teeth S5 16 Without fitting S 470 [17.91] 2 charge pumps SAE A; 9 teeth Α 475,8 [18.73]

Rear

SAE BB; 15 teeth

**S4** 

Gear pumps are always delivered flanged on the axial pump. They can not be sold alone.

With

11

or

16

Ε

В

С

475,8 [18.73]

479 [18.86]

479 [18.86]

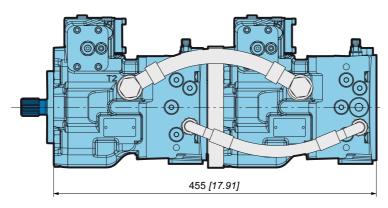
SAE A; 11 teeth

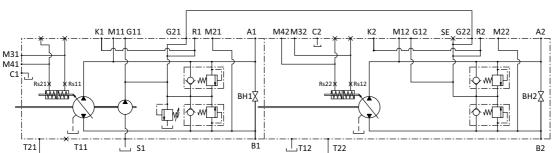
SAE B; 13 teeth

SAE BB; 15 teeth

### Example of tandem configuration with 1 charge pump









Ports T and G of the first pump must be connected with ports T and G of the second pump.



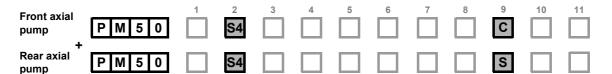
Model

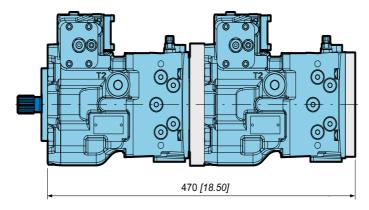
Technical specifications

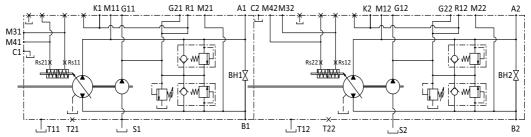
Operating Parameters

System design Parameters

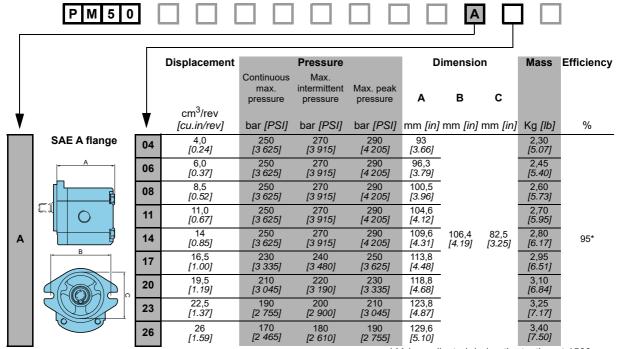
### Example of tandem configuration with 2 charge pumps







### **Gear pumps**



\* Value collected during the testing at 1500 rpm



It's possible to provide different gear pump (group 3 with SAE B pad).



Technical specifications

Operating Parameters

System design Parameters

**CONTROLS** 

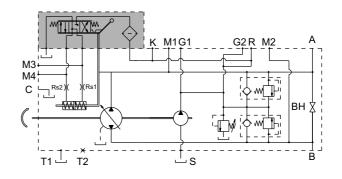
### Mechanical servo control with feedback

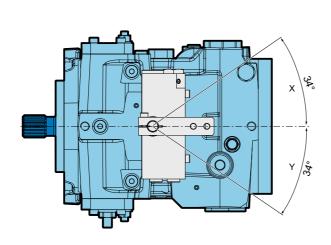
	1	2	3	4	5	6	7	8	9	10	11
P M 5 0			Α								

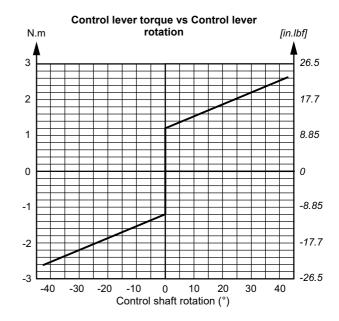
Control function	The variation in pump displacement is reached by control lever rotation to adjust hydraulic servo piston position. Control lever range is 40°. Movement of control lever is independent of the pressure and pump speed.
Control regulation	To avoid sudden accelerations and stoppages, two restrictors (Rs1 and Rs2) are inserted between servo control and hydraulic servo piston. They are used to regulate control shifting speed.
Feedback function	The feedback system between swash plate and hydraulic servo piston permit to maintain costant displacement of the pump if the pressure between pump and hydraulic motor changes.  The feedback function is reached by a lever that connects the swashplate and the hydraulic servo piston.

### Flow rate determination

Rotation	Control	Output	Input
Clockwise (R)	Х	Α	В
Ciockwise (K)	Υ	В	Α
Counter clockwise (L)	Χ	В	Α
Counter Clockwise (L)	Y	Α	В









The spring return feature in the control unit is not a safety device.

Features

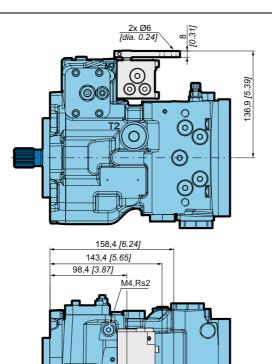
Controls

Options

22/04/2024

29

### Dimensions with control A



See page 7 for other dimensions and page 6 for port characteristics.



See option MI (page 42) to add neutral position switch.



### Hydraulic servo control

	1	2	3	4	5	6	7	8	9	10	11
P M 5 0			S								

Control function	The variation in pump displacement is reached by pressure adjustment on the M3 and M4 servo control ports. These ports are controlled by hydraulic proportional joystick (containing pressure reduction valves).  The joystick supply can by obtained by taking pressure from the auxiliary pump (R connection). Basic joystick can be provided upon request.
Control regulation	The servo control response time can be adjusted by two restrictors (Rs1 and Rs2) inserted on the joystick supply line (from 0,6 to 1,2 mm [from 0.02 to 0.05 in]). The servo control operation pressure curve in both control directions goes from 4,5 to 15 bar [from 65 to 218 PSI].  The adjustment curve of the hydraulic control system has to be wider, from 4 to 16 bar [from 58 to 232 PSI].



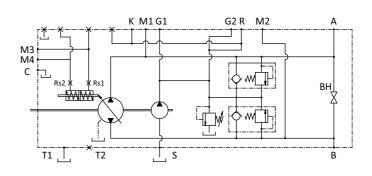
Other curves can be used in relation to valve plate timing. Contact your Poclain Hydraulics application engineer for further info.



For the selection of the regulation curve (with or without step) of the Joystick contact your Poclain Hydraulics application engineer.

### Flow rate determination

Rotation	М3	M4
Clockwise (R)	Α	В
Counter clockwise (L)	В	Α



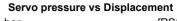
The spring return feature in the control unit is not a safety device.

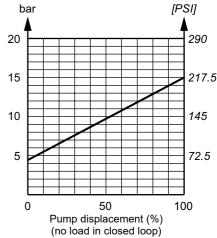


Hydraulic joystick can be with or without step.



The back pressure of the return line of the joystick and the drive line of the pump has an influence on Servo pressure vs Displacement values.







Above graph is just an example that shows the relationship between servo pressure and displacement.

Mod

Technical specifications

**Operating** 

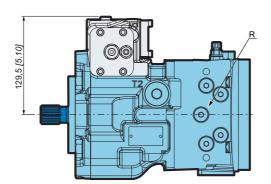
System design Parameters

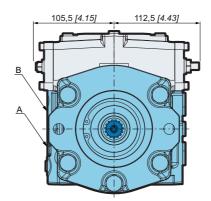
Features

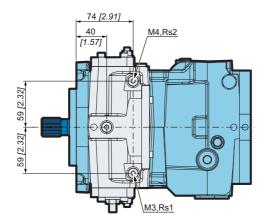
Controls

Options

### Dimensions with control S





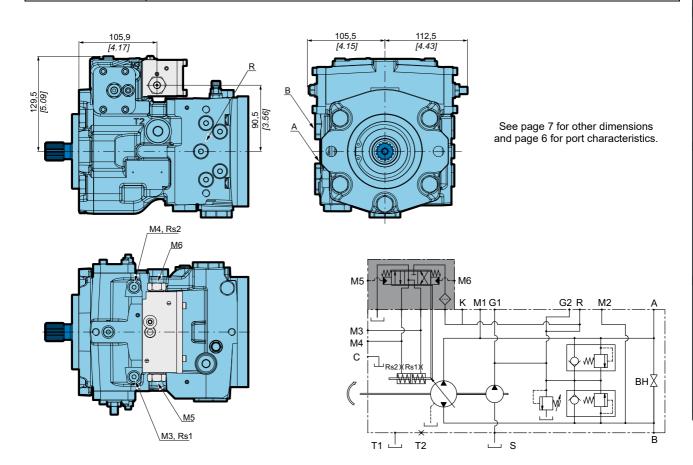


See page 7 for other dimensions and page 6 for port characteristics.

### Hydraulic servo control with feedback

	1	2	3	4	5	6	7	8	9	10	11
P M 5 0			T								

Control function	The variation in pump displacement is reached by pressure adjustment on the M5 and M6 feedback control ports.  These ports are controlled by hydraulic proportional joystick (containing pressure reduction valves). The joystick supply can by obtained by taking pressure from the auxiliary pump (R connection). Basic joystick can be provided upon request.
Control regulation	The servo control operation curve in both directions goes from 6 to 15 bar [from 87 to 218 PSI]. The adjustment curve of the hydraulic control system has to be wider, from 5 to 16 bar [from 73 to 232 PSI].
Feedback function	The feedback system between swash plate and hydraulic servo piston permit to maintain constant displacement of the pump if the pressure between pump and hydraulic motor changes.  The feedback function is reached by a lever that connects the swashplate and the hydraulic servo piston. To avoid sudden accelerations and stoppages, two restrictors (Rs1 and Rs2) are inserted between the servo control and the hydraulic servo piston.





Contact your Poclain Hydraulics application engineer in case of special needs of the control.



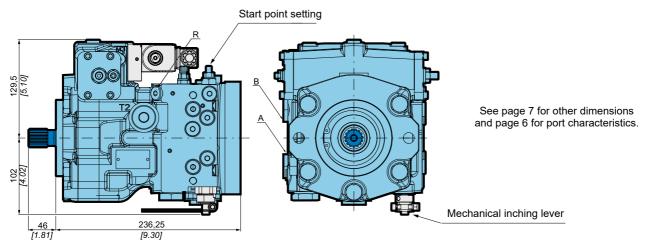
Hydraulic joystick can be with or without step.

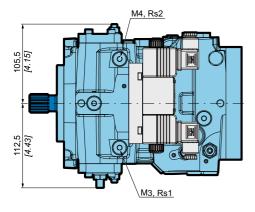
### **Hydraulic automotive control**

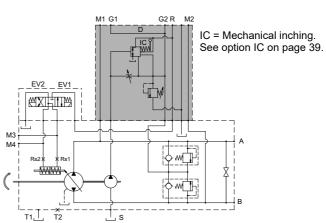


Control function	The variation in pump displacement is reached by continuous electro-hydraulic valve adjustment. The adjustment is precised by pilot pressure controlled by solenoid control. The pilot pressure increases proportionally to the rotation of the pump. The pump displacement increases corresponding to the higher pilot pressure.
	In case the engine is overloaded, the rotation rate decreases and the pilot pressure is reduced
Control regulation	causing a pump displacement reduction with a corresponding drop in absorbed power.









### Flow rate determination

Rotation	Pressure	Output	Input
Clockwise (R)	EV1	В	Α
Ciockwise (ix)	EV2	Α	В
Counter	EV1	Α	В
clockwise (L)	EV2	В	Α



The power and torque curve of the engine are necessary for automotive valve setting.



IC rotation angle controls pump destroke. Angle of regulation is 25°.

Model

Technical specifications

### **Electrical on-off servo control**

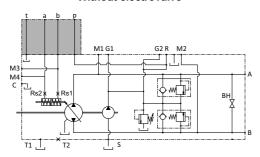
### Control with return spring

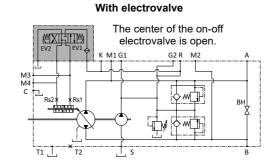


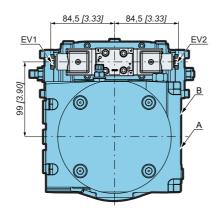
Control function	closed CETOP 2 connection. If the electrovalve motion is stopped, the pump goes back to neutral position due to the hydraulic servo piston return springs.
Control regulation	The displacement reached is defined by the starting time of the electrovalve and by diameter of restrictors (Rs1 and Rs2) inserted between the electrovalve and the hydraulic servo piston. The pump can be supplied either without electrovalve (B00) or with electrovalve (B12 / B24).

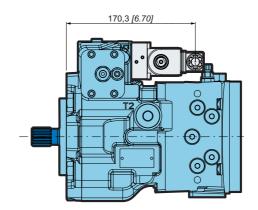
Supply voltage	
Without	B00
12V	B12
24V	B24

### Without electrovalve



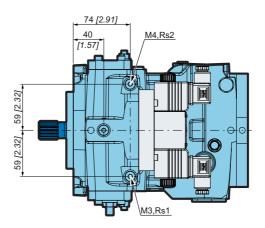






See page 7 for other dimensions and page 6 for port characteristics.

Solenoids specification						
Operating voltage	12 VDC ± 10%	24 VDC ± 10%				
Resistance at 20°C [68°F]	$5,3~\Omega\pm7\%$	21,2 Ω ± 7%				
Connector type	DIN 43650					
Nominal power 27 W						
Protection		65				
Mass 0,215 kg [0.47 lb]						



### **Electro-proportional servo control**



Control function	The variation in pump displacement is reached by current adjustment applied to proportional valve coils. The coils then adjust the pressure of the servo control connected to the hydraulic servo piston. The flow rate direction depends on activated coil.
Control regulation	The reaction time can be controlled by ramps installed on the card and by restrictors (Rs1 and Rs2) positioned between the electrovalves and the hydraulic servo piston.
Automotive function	Electro-proportional servo control combined with ECU unit and appropriate software can be used for Higher performances Automotive control.



### Flow rate determination

Rotation	EV1	EV2
Clockwise (R)	Α	В
Counter clockwise (L)	В	Α



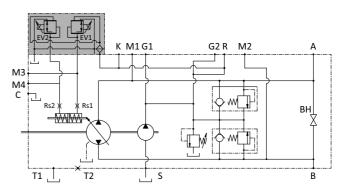
Valve plate timing and regulation curve of proportional valve influence the flow.
Contact your Poclain Hydraulics application engineer for further info.



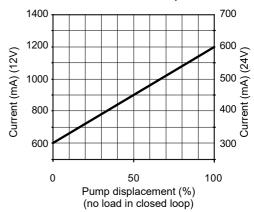
The current must not exceed 1500 mA under 12V and 800 mA under 24V.



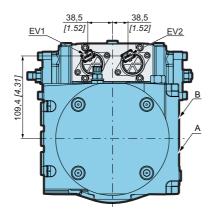
The spring feature in the control unit is not a safety device.

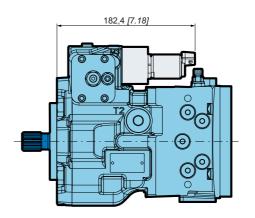


### **Electrovalve current vs Displacement**

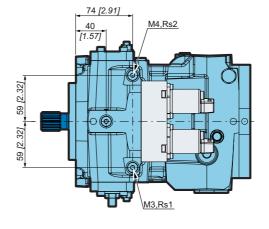


#### Dimensions with control P





See page 7 for other dimensions and page 6 for port characteristics.



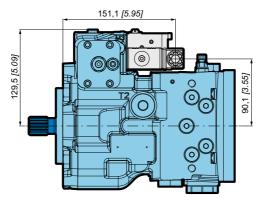
Solenoids specification		
Operating voltage	12 VDC	24 VDC
Current	1500 mA	750 mA
Resistance at 20°C [68°F]	5,3 Ω ± 5%	21,2 Ω ± 5%
Connector type	AMP Junior Timer, Deutsch DT04-2P	
Protection	IP6K6	/ IPX9K

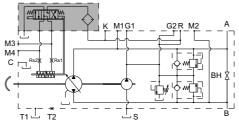
## Electro-proportional servo control with feedback

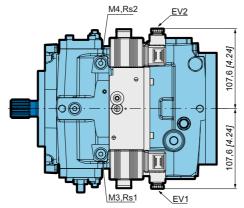


Control function	The variation in pump displacement is reached by current adjustment applied to electro- proportional coils. The coils then adjust the pressure of the servo control. The flow rate direction depends on activated coil.
Control regulation	The reaction time can be controlled by ramps installed on the card and by restrictors (Rs1 and Rs2) inserted between the servo control and the hydraulic servo piston.
Feedback function	The feedback function is reached by a lever that connects the swashplate and the hydraulic servo piston. To avoid sudden accelerations and stoppages, two restrictors (Rs1 and Rs2) are inserted between the servo control and the hydraulic servo piston.

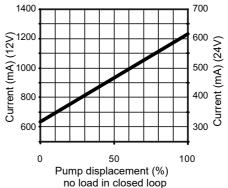








# Electrovalve current vs Displacement



See page 7 for other dimensions and page 6 for port characteristics.

Solenoids specification		
Operating voltage	12 VDC	24 VDC
Current	1500 mA	750 mA
Resistance at 20°C [68°F]	5,3 Ω ± 5%	21,2 Ω ± 5%
Connector type	Hirschman DIN 43650 Deutsch DT04-2P AMP Junior Timer (standard for PMe)	
Protection	IP6K6 / IPX9K	

Roller bearing

CR



## **OPTIONS**

The PM50 can be provided with high capacity roller bearing to extend lifetime of the application. According to characteristics of shaft load, the duty cycle and lifetime epectancy a roller bearing might be needed.

Consult your Poclain Hydraulics application engineer.

## **Customized identification plate**

P M 5 0 DP

The PM50 can be provided with customized identification plate (customer part number engraved on the plate).



This option is available only for minimum volume of 50 pieces.



Consult your Poclain Hydraulics application engineer for other possibilities.

#### **Mechanical inching**

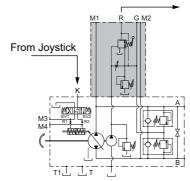
P M 5 0 1 2 or D12 4 5 6 7 8 9 10 11 IC

The PM50 with Hydraulic automotive control D (page 34) can be provided with an Inching lever to reduce the pilot pressure independently of the pump rotation speed

#### **Hydraulic inching**

PM 5 0 1 2 D12 4 5 6 7 8 9 10 11 HI

The PM50 with Hydraulic automotive control D (page 34) can be provided with a pressure reducer valve (connected with port K). Its function is to reduce the displacement of pump. The pedals type VB3-002 (only inching function) or VB3-012 (inching and service brake function) can be provided upon request.



Model

Technical specifications

**Operating** 

System design

Features

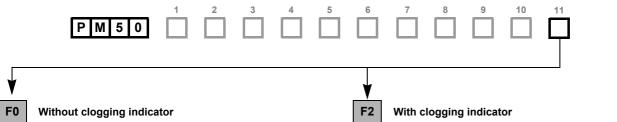
Controls

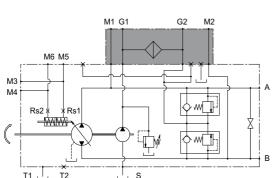
Options

#### Filter on pressure line

The PM/PMe50 can be provided with a F0/F2 filter. It's placement on pressure line ensures that only filtered oil enters the pump closed loop. Maximum pressure difference between filter cartridge input and output is 2 bar [29 PSI]. After reaching 2 bar [29 PSI], the cartridge has to be changed.

Tightening torque: 35 Nm [309 in.lbf]. Max. working pressure: 30 bar [435 PSI]. Filter fitness is of 10 micron.





T1 上 T2	∟ S
Clogging indicator s	pecification
Differential working adjustment	3 ± 0,2 bar [44 ± 3 PSI]
Working temperature	-30°C ~ 110 °C [-22°F ~ 230°F]
Max. vibration level	50 g
Connector type	AMP super seal,

Port	Function	UNF ISO 11926-1	GAS ISO 1179-1
G1/G2	Auxiliary/Charge pressure	9/16-18 UNF-2B	13G-G1/4
M1/M2	A/B pressure	9/16-18 UNF-2B	13G-G1/8

2 way

0,1-0,2 A max.

Normally closed contact. Thread of the clogging indicator is internally connected to ground.



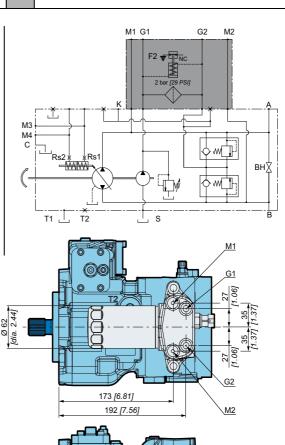


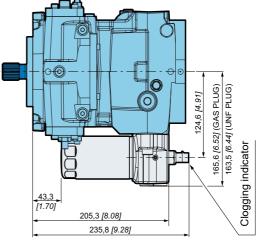
Current range

This option is not compatible with option SS (Speed



In case of tandem pump use, each pump must be equipped with it's own filter and charge pump.



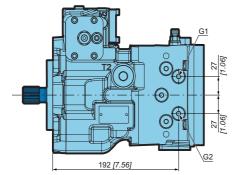


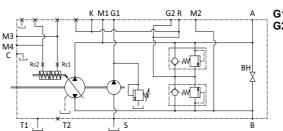
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## **External connections for filter**



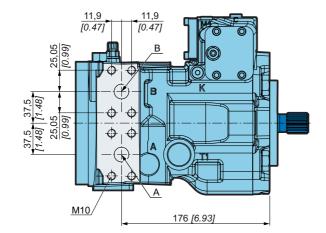




G1 = to filter on line G2 = Return from filter on line

## **SAE flange ports**

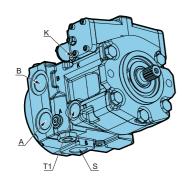


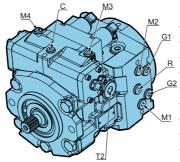


Port	Function	SAE flange
A-B	Services	PN400-DN19
A-D	Services	4xM10
С	Case pressure	13G-G1/4
G1/G2	Auxiliary/Charge pressure	13G-G1/4
M1/M2	A/B pressure	10G-G1/8
M3/M4	Servo control	13G-G1/4
K	External servo pilot	10G-G1/4
R	Servo pilot pressure	13G-G1/4
S	Suction	34G-G1
T1/T2	Drain	27G-G3/4

## **UNF** threads ports







	Port	Function	UNF ISO 11926-1
1	A/B	Services	1"5/16-12 UNF-2B
	С	Case pressure	9/16-18 UNF-2B
R	G1/G2	Auxiliary/Charge pressure	9/16-18 UNF-2B
<u>G2</u>	M1/M2	A/B pressure	9/16-18 UNF-2B
	M3/M4	Servo control	9/16-18 UNF-2B
1	K	External servo pilot	7/16-20 UNF-2B
-	R	Servo pilot pressure	9/16-18 UNF-2B
	S	Suction	1"5/16-12 UNF-2B
	T1/T2	Drain	1"1/16-12 UNF-2B



Special fittings needed to ensure compatibility with option SS (Speed sensor T4). Consult your Poclain Hydraulics application engineer.

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ΜÕ

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Controls

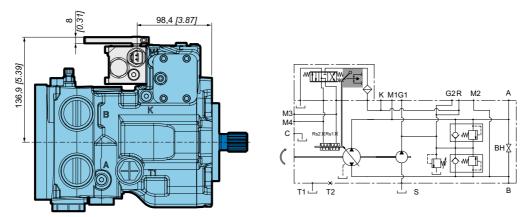
UNF

Options

## **Neutral position switch**



The PM50 with Mechanical servo control A (page 29) can be provided with a micro switch to avoid engine start in case the control lever is not centered (zero position).

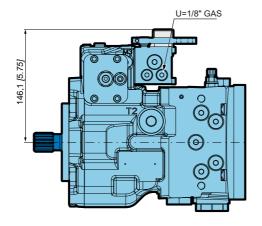


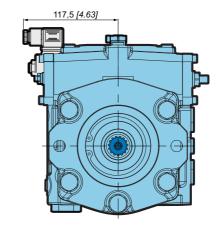
Electrical characteristics		
Type of connector	Deutsch DT04-2P	
Output	NC and NO	
Cable connections	PG 13,2	
Max. current	10 A	
Electric load type	Resistive	
Operating temperature	from -25°C to 80°C [-13°F to 176°F]	
Type of protection	IP 67	

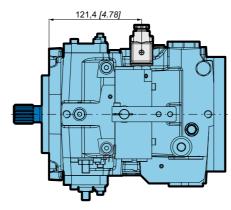
## Safety valve



The PM50 pump with Mechanical servo control A (page 29) can be provided with a safety valve VPU. Without current, the VPU disconnects the servo control from the charge pressure and engages negative brake.







M3  C  Res  Res  T1  T2  S	G2R M2 — A

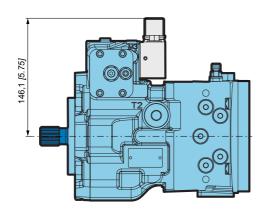
Coil specification	
Type of connector	DIN 43650
Nominal voltage	12V DC
Power	18W
Type of protection	IP 65
Ambient temperature range	from -30°C to 60°C [-22°F to 140°F]
Magnet wire insulation	Class H -> 200°C [392°F]
Heat insulation	Class H -> 180°C [356°F]
Mass	0,19 kg <i>[0.42 lb]</i>
Lead wires	600V rating with strain relief

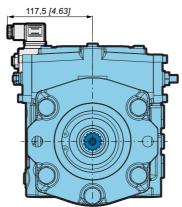
Connector specification		
AC rated voltage	250V max.	
DC rated voltage	300V max.	
Pin contact rated flow	10A	
Pin contact max. flow	16A	
Max. cable section	1,5 mm² [0.002 in²]	
Ø Cable gland PG09-M16x1,5	6 to 8 mm [0.24 to 0.31 in]	
Type of protection	IP65 EN60529	
Insulation class	VDE 0110-1/89	
Operating temperature	from -40°C to 90°C [-40°F to 194°F]	

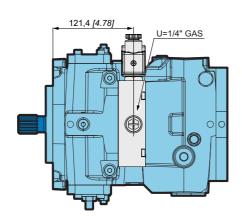
## Safety valve

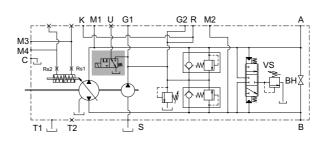


The PM50 pump with Hydraulic servo control S (page 31) can be provided with a safety valve VPU. Without current, the VPU disconnects the servo control from the charge pressure and engages negative brake.









Coil specification	
Type of connector	DIN 43650
Nominal voltage	12V DC
Power	18W
Type of protection	IP 65
Ambient temperature range	from -30°C to 60°C [-22°F to 140°F]
Magnet wire insulation	Class H -> 200°C [392°F]
Heat insulation	Class H -> 180°C [356°F]
Mass	0,19 kg <i>[0.42 lb]</i>
Lead wires	600V rating with strain relief

Connector specification	
AC rated voltage	250V max.
DC rated voltage	300V max.
Pin contact rated flow	10A
Pin contact max. flow	16A
Max. cable section	1,5 mm² [0.002 in²]
Ø Cable gland PG09-M16x1,5	6 to 8 mm [0.24 to 0.31 in]
Type of protection	IP65 EN60529
Insulation class	VDE 0110-1/89
Operating temperature	from -40°C to 90°C [-40°F to 194°F]

## Finishing coat



The PM50 can be delivered with finishing coat when requested. Standard paint is RAL 9005 (black color).

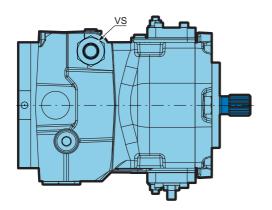


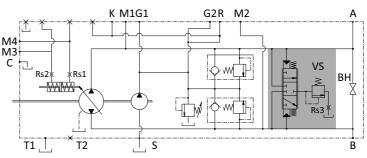
Consult your Poclain Hydraulics application engineer for other colors of topcoat.

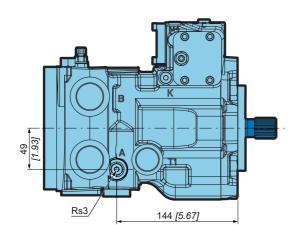
## Flushing valve



The PM50 can be provided with a flushing valve VS to discharge the oil inside the pump casing through a relief valve of the flushing valve. The exchange valve is useful in case the temperature of the oil in the closed circuit is too high.







#### Flushing flow L/min [gal/min]

		Ornice diameter RS3 mm [m]							
		1,4 [0.055]	1,8 [0.071]	2,2 [0.087]					
Delta	20 [290]	2,8 [0.75]	4,5 [1.19]	5,5 <i>[1.46]</i>					
pressure bar [PSI]	25 [363]	3,6 [0.96]	5,9 [1.55]	7,2 [1.90]					
	30 [435]	4.3 [1.13]	7.0 [1.85]	8.5 [2.26]					



Refer to Poclain Hydraulics service manual for info about restrictor Rs3 exchange.

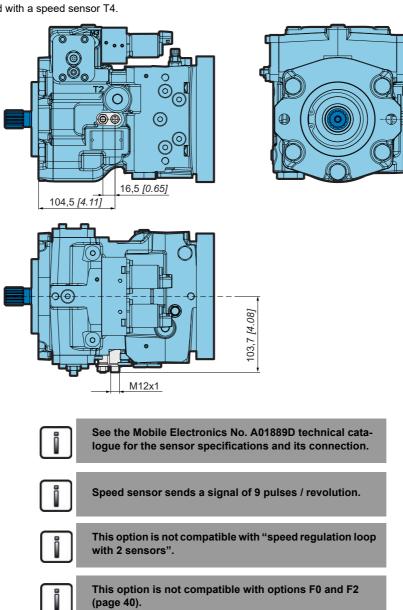


Contact you Poclain Hydraulics application engineer for restrictor Rs3 selection.

#### **Speed sensor**



The PM50 can be provided with a speed sensor T4.



## Fluorinated elastomer seals

1	1 2	3	4	5	6	7	8	9	10	11
P M 5 0	] [									EV

The PM50 can be provided with fluorinated elastomer seals. Standard NBR sealing are designed to resist to temperature up to 90°C [194°F] and to HV type oils. If your application is outside these limits, fluorinated elastomer seals might be needed.

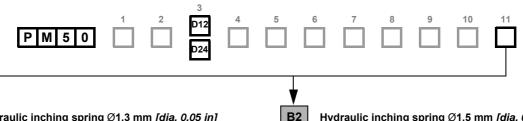


For application of this option please contact your Poclain Hydraulics application engineer.

#### **Brake inching**

**B1** 

The PM50 with Hydraulic automotive control D (page 34) can be provided with the B1/B2 brake inching. Its function is to reduce the displacement of pump. Reduction is achivied via brake pedal (connected to Z port). The pedal type VB3-010 (only inching function or inching and sevice brake function) can be provided upon request.

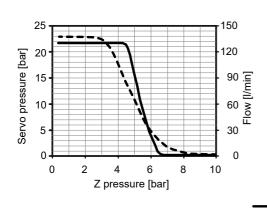


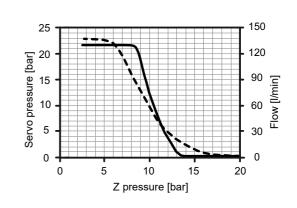
Flow

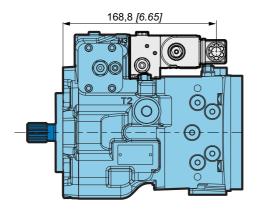
- Servo pressure

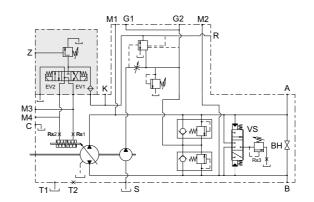
Hydraulic inching spring Ø1,3 mm [dia. 0.05 in]

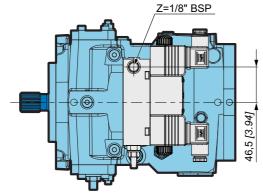
Hydraulic inching spring Ø1,5 mm [dia. 0.06 in]











Solenoids specification								
Operating voltage	12 VDC ± 10%	24 VDC ± 10%						
Current	1500 mA	750 mA						
Resistance at 20°C [68°F]	$5,3~\Omega\pm7\%$	21,2 Ω ± 7%						
Connector type	DIN 43650							
Power	27 W							
Type of protection	IP65							
Mass	0,215 kg <i>[0.47 lb]</i>							



The hydraulic inching valve B1/B2 does not provide any sealing between closed loop circuit and pilot circuit. When choosing this function please be sure that oil to pilot the inching is coming from the same tank as the closed loop.

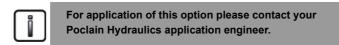
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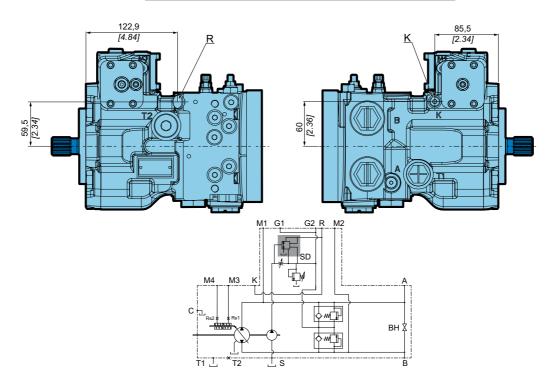
#### **Anti-stall valve**

	1	2	3	4	5	6	7	8	9	10	11
P M 5 0											SD

The PM50 can be provided with anti-stall valve SD. It consists a cartridge valve (same cartridge valve as automotive control) which provides a pressure signal for the servo piston of the pump related to the speed of engine.

Its function is to reduce pressure for servo piston in case of engine overload and consequent rpm reduction. As a result the pump destrokes with an anti-stall effect.







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